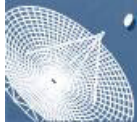


KINEMATICS AND DYNAMICS OF PARALLEL STRUCTURES FOR CONTROL DESIGN

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KINEMATICS AND DYNAMICS OF PARALLEL STRUCTURES FOR CONTROL DESIGN

Brief outline:


-  → **Introduction of Parallel Structures**
-  → **Kinematic Model**
-  → **Dynamic Model Including the Model of DC Motor**
-  → **Design and Examples of Control Approaches**
-  → **Simulation Samples of Actuators**
-  → **Conclusion**

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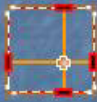
Introduction of Parallel Structures

serial structures



↓ stiffness
↓ dynamics

adequately actuated


□ **problems**

↓ existence of singularities

! redundant drives are not fully determined and their mutual fighting may happen !

parallel structures

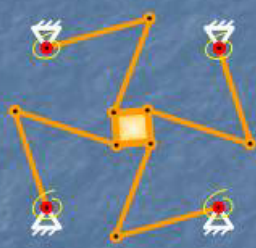

□ **idea**



↑ dynamics
↑ stiffness

actuated redundantly

enable to solve additional conditions ↑

□ **examples**

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KINEMATICS AND DYNAMICS OF PARALLEL STRUCTURES FOR CONTROL DESIGN

Kinematic Model

The definition of coordinates:

- drive z (φ_i)
- operational q (x_{C1}, y_{C1}, ψ)
- and other auxiliary z_a (ψ_i)

All together called physical:

- independent (adequate to DOF)
- or dependent (remaining ones)

Interrelated by: $f(q, z, z_a) = 0$

6 component equations
in directions x and y
transformed to the relations only among z and q

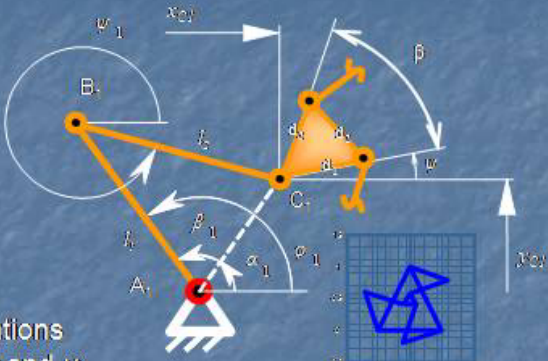
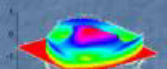
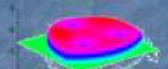
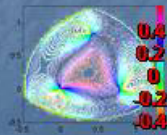

i.e. for positions: $f(q, z) = 0$

velocities:

$$\frac{\partial f(q, z)}{\partial q} \dot{q} + \frac{\partial f(q, z)}{\partial z} \dot{z} = 0 \Rightarrow \Phi_q(q, z) \dot{q} + \Phi_z(q, z) \dot{z} = 0$$

accelerations:

$$\dot{\Phi}_q(q, z) \dot{q} + \Phi_q(q, z) \ddot{q} + \dot{\Phi}_z(q, z) \dot{z} + \Phi_z(q, z) \ddot{z} = 0$$

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KINEMATICS AND DYNAMICS OF PARALLEL STRUCTURES FOR CONTROL DESIGN



Dynamic Model Including the Model of DC Motor



- DAE

$$\mathbf{M}\ddot{\mathbf{s}} = \mathbf{g} + \mathbf{T}\mathbf{u} + \Phi^T\boldsymbol{\lambda} \Leftrightarrow 19 \text{ equations with 19 unknowns}$$

used e.g. for determination of passive force effects

$$\mathbf{f}(\mathbf{s}(t)) = 0$$

- ODE

$$\mathbf{R}^T\mathbf{M}\mathbf{R}\ddot{\mathbf{q}} + \mathbf{R}^T\mathbf{M}\dot{\mathbf{R}}\dot{\mathbf{q}} = \mathbf{R}^T\mathbf{g} + \mathbf{R}^T\mathbf{T}\mathbf{u} \Leftrightarrow 3 \text{ equations with 3 unknowns}$$

($\mathbf{F}\mathbf{M} = \mathbf{R}^T\mathbf{T}\mathbf{u}$)

- State model

$$\dot{\mathbf{X}} = \mathbf{f}(\mathbf{X}) + \mathbf{g}(\mathbf{X})\mathbf{u} \Leftrightarrow 6 \text{ equations with 6 unknowns}$$

used for fast simulation

$$\mathbf{y} = \mathbf{C}\mathbf{X}$$

- Linearized

discrete state model

$$\mathbf{X}(k+1) = \mathbf{A}\mathbf{X}(k) + \mathbf{B}\mathbf{u}(k) \Leftrightarrow 6 \text{ equations with 6 unknowns}$$

used for control design

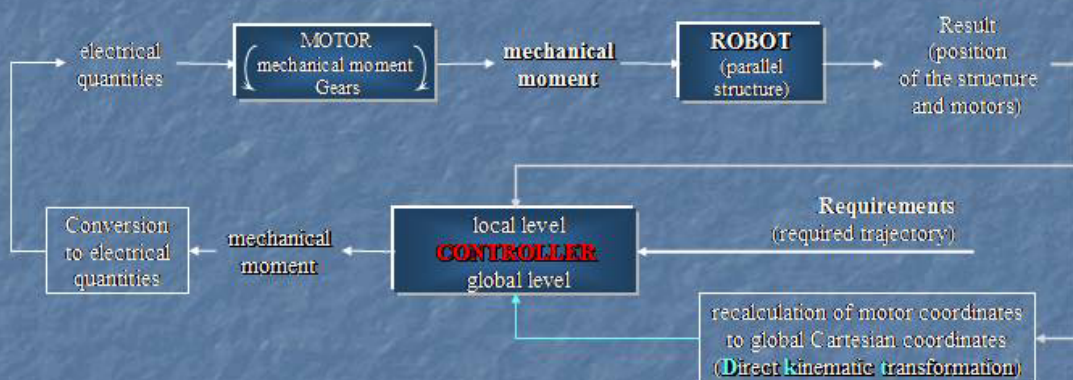
$$\mathbf{y}(k) = \mathbf{C}\mathbf{X}(k)$$

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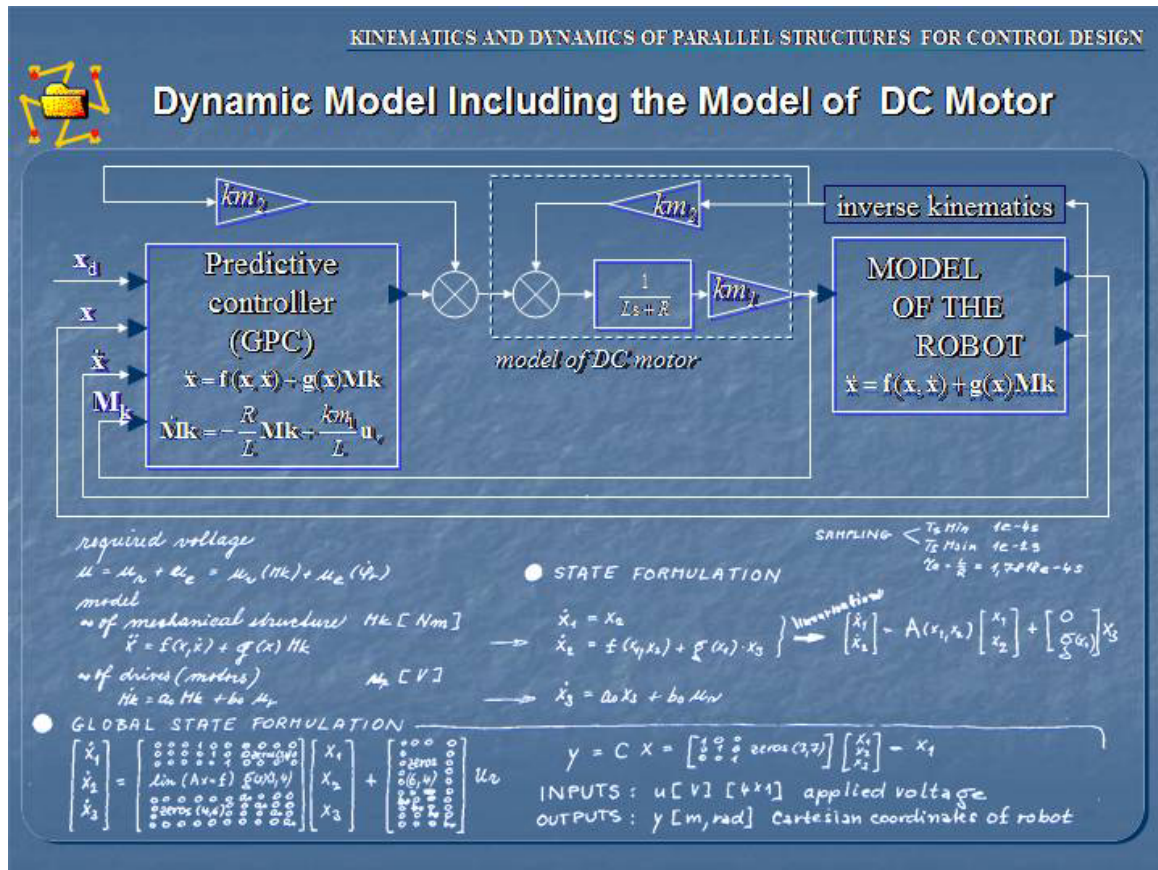
KINEMATICS AND DYNAMICS OF PARALLEL STRUCTURES FOR CONTROL DESIGN



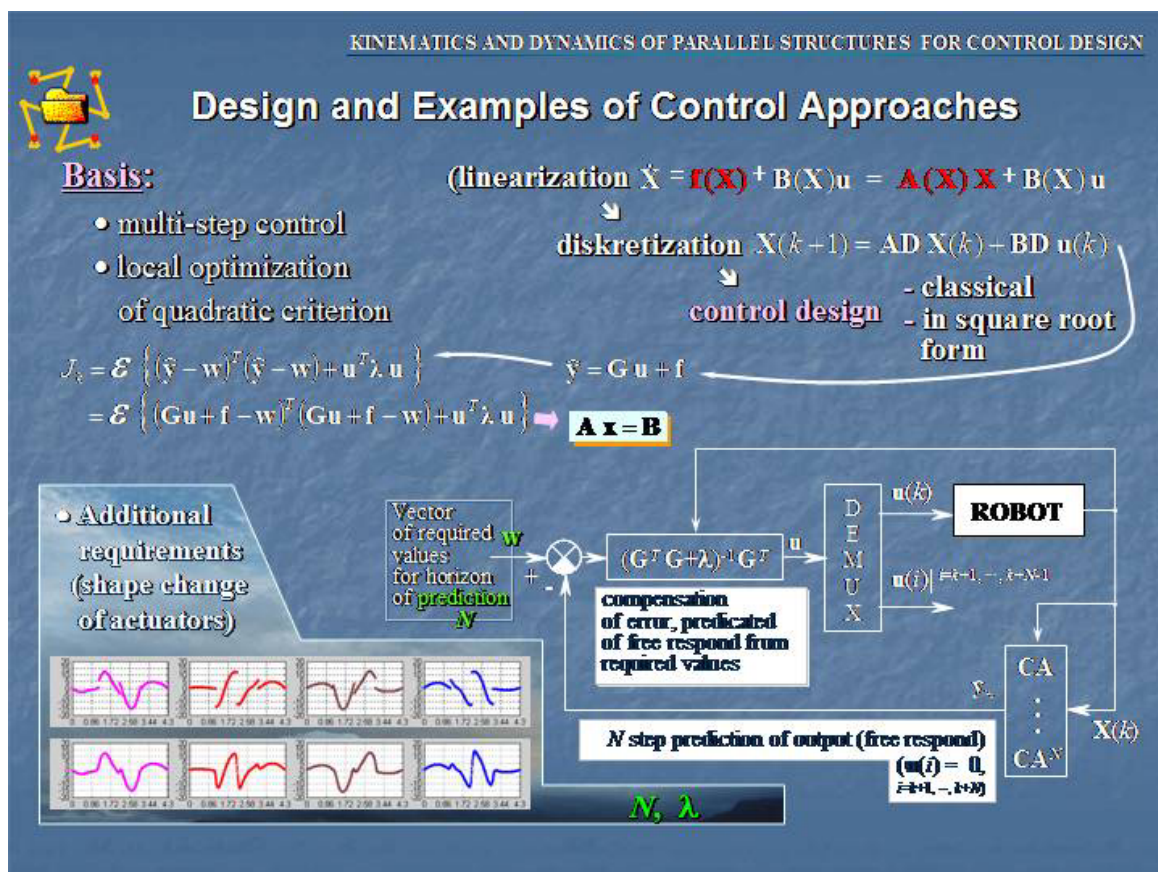
Dynamic Model Including the Model of DC Motor



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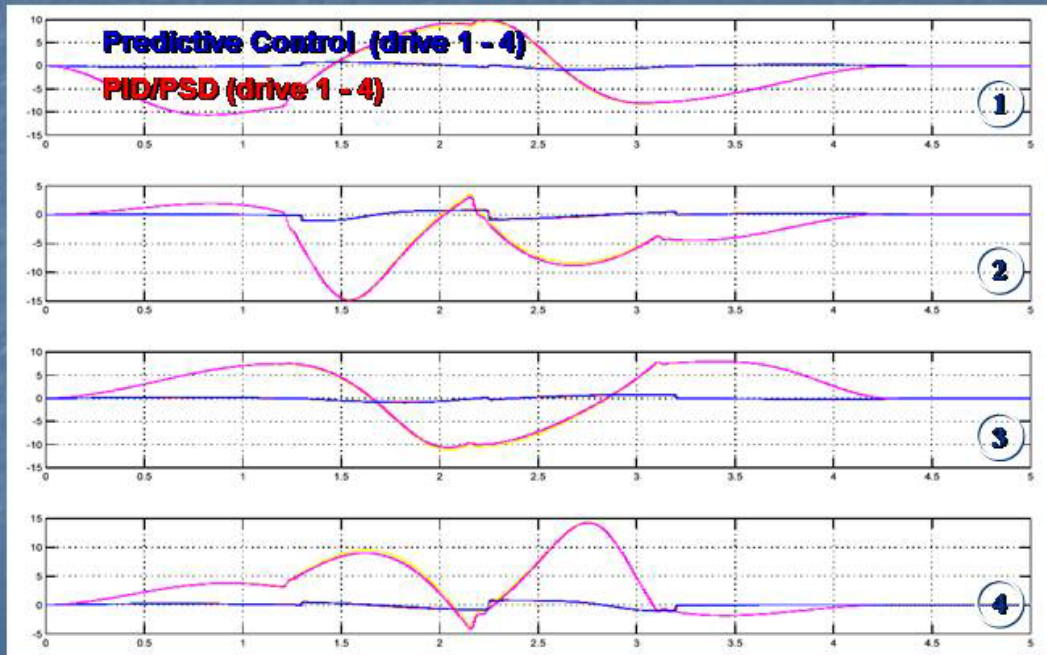


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KINEMATICS AND DYNAMICS OF PARALLEL STRUCTURES FOR CONTROL DESIGN



Simulation Samples of Actuators



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KINEMATICS AND DYNAMICS OF PARALLEL STRUCTURES FOR CONTROL DESIGN



Conclusion I. - Modelling

Classification of singular cases

- Inverse Kinematics Singularities**
 some group of arms is in a fully stretched-out or folded-back configuration; the structure loses DOF according to the configuration; in these configurations, the changes of working elements (external arms) do not cause any change of movable platform $\det(\Phi_2) = 0$.
- Direct Kinematics Singularities**
 the directions of internal arms intersect in one point of movable platform; (the platform can slightly rotate in spite of locked drives);
 or the directions are parallel to one another (the platform can perform infinitesimal translations perpendicular to the parallel arms); $\det(\Phi_1) = 0$.
- Combined Singularities** $\det(\Phi_1) = 0$ and $\det(\Phi_2) = 0$;
 it is caused by dependence of structural relations (geometrical constraints).

Diagrams illustrate the configurations for Inverse Kinematics Singularities (fully stretched-out or folded-back arms) and Direct Kinematics Singularities (intersecting or parallel arm directions). The combined singularities are shown as regions where the determinants of the Jacobian matrices are zero.

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KINEMATICS AND DYNAMICS OF PARALLEL STRUCTURES FOR CONTROL DESIGN



Conclusion II. - Control



Model based approaches generate optimized actuators



Model based approaches provide additional requirements



Model based approaches need relatively accurate model



Model based approaches are time consuming way of control



END